




# Characterization of the lateral gas–solid flow in a circulating fluidized bed boiler using simple mathematical model based on CPFD simulation data

Manxia Shang<sup>a,b</sup>, Ling Jiang<sup>a</sup>, Li Zhao<sup>b</sup>, Zhong Huang<sup>a,b,c</sup>, Xiwei Ke<sup>b,c,\*</sup> , Junfu Lyu<sup>a,b,c,\*</sup>

<sup>a</sup> Key Laboratory for Thermal Science and Power Engineering of Ministry of Education, Tsinghua University, Beijing 100084, China

<sup>b</sup> Shanxi Research Institute of Huairou Laboratory, Taiyuan 030006, China

<sup>c</sup> Beijing Huairou Laboratory, Beijing 101499, China

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## ABSTRACT

Operation safety, combustion efficiency, and pollutant emissions of circulating fluidized bed (CFB) boilers are significantly affected by the lateral gas–solid flow uniformity, which can be featured by the voidage ( $\varepsilon_g$ ) as well as particle lateral movement velocity ( $u_p$ ). To obtain abundant data to support quantitative analysis and modeling, the gas–solid flow details in a 170 t/h CFB boiler were numerically simulated using the computational particle fluid dynamics (CPFD) method, which has been validated by field test data. Based on the original simulation data set, simple models for the prediction of lateral profiles of  $\varepsilon_g$  and  $u_p$  were developed. In addition, both theoretical model and genetic algorithm-back propagation (GA-BP) neural network were applied to predict the lateral solids movement behavior. Results show that both two models exhibit excellent prediction accuracy with the coefficient of determination ( $R_2$ ) exceeding 0.99. While the theoretical model derived from physical equations is relatively simple and has a clear physical meaning. All the simple models developed in this paper can be embedded into the overall CFB mathematical model framework, facilitating the comprehensive analysis of the operational characteristics for large-scale industrial CFB boilers.

## 1. Introduction

Circulating fluidized bed (CFB) combustion technology has seen widespread development in recent years due to its excellent fuel adaptability (e.g., low-carbon fuel blending such as biomass and solid waste), pollutant control capabilities, and load regulation characteristics. It has also gradually evolved toward higher parameters and larger capacities (Błaszczuk and Nowak, 2015; Krzywanski et al., 2015; Nowak, 2003; Cai et al., 2018).

However, as furnace size increases, the issue of lateral non-uniformity in CFB boilers has become more pronounced. CFB boilers are typically fed with fuel on one side. This inherent asymmetry in the structural layout leads to uneven fuel distribution, causing inhomogeneous heat transfer and combustion within different regions. As a result, issues such as heating surface wear and temperature deviation may arise (Song et al., 2018). Although the secondary air is mostly designed to be symmetrically fed into the furnace, the direction of the furnace depth is often up to several meters long, and due to the limited penetration depth

of the secondary air, there will be some dead zones. Poor design can severely disrupt the internal circulation of solids at the furnace's bottom, further exacerbating the inhomogeneity of solids distribution (Koeninger et al., 2017; Namkung and Sang, 2000). Furthermore, in the context of the global energy transition and the future large-scale integration of renewable energy into the power grid, CFB boilers will often operate in a low-load state (Shang et al., 2024). Nevertheless, when the load is significantly reduced, the lateral non-uniformity will be further exacerbated due to the reduction of wind speed and circulating flow rate (Ke et al., 2022). The lateral non-uniformity directly impacts the operation safety, combustion efficiency, and pollutant emissions of CFB boilers (Krzywanski et al., 2015; Jin et al., 2021; Bayham et al., 2017; Scala, 2018). To ensure the safe and economical operation of CFB boilers, it is necessary to conduct in-depth research on the lateral non-uniformity.

Voidage ( $\varepsilon_g$ ) and particle lateral movement velocity ( $u_p$ ) are key parameters in studying of lateral uniformity. The former serves as the basis for CFB fluid flow and combustion calculations (Xu et al., 1999),

\* Corresponding authors at: Shanxi Research Institute of Huairou Laboratory, Taiyuan 030006, China.

E-mail addresses: [kexiwei@sxri.hrl.ac.cn](mailto:kexiwei@sxri.hrl.ac.cn) (X. Ke), [lvjf@mail.tsinghua.edu.cn](mailto:lvjf@mail.tsinghua.edu.cn) (J. Lyu).

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**Table 1**  
Measurements of the lateral voidage in circulating fluidized bed systems.

Authors	$D_t$ (mm)	$H_t$ (m)	$D_p$ ( $\mu\text{m}$ )	$\rho_p$ (kg/m <sup>3</sup> )	Techniques
Weinstein et al., 1985	152	8.5	59	1460	X-ray cameraabsorptiometry
Hartge et al., 1988	400	8.4	85; 120	1500; 2600	Fiber optical probe
Tung et al., 1988	90	10.0	54	929	Fiber optical probe
Herb et al., 1989	150	10.0	68	1500	Capacitance probe
Zhang et al., 1991	32; 90; 300	2.8; 10.0; 12	54; 75; 42.8	929; 607; 2003	Fiber optical probe
Xu et al., 1999	90	11	54	929.5	Fiber optical probe
Xu et al., 2004	90	11	54	929.5	Fiber optical probe
Zhu et al., 2013	100	7.9	90	1500	Fiber optical probe
Hensler et al., 2016	190	11.3	69.1	1027	X-ray computed tomography
Weber et al., 2018	305	15.3	886	863	Electrical capacitance volume tomography
Xu et al., 2018	384	60	200	2660	Sampling tube

while the latter governs the dispersion characteristics of gas–solid and the lateral mass exchange flow rates in CFB (Farzaneh et al., 2011).

For the lateral voidage distribution, researchers have conducted extensive measurements (Table 1) and characterizations (Table 2) in fluidized bed systems. As early as 1986, Weinstein et al. (Weinstein et al., 1985) measured the lateral voidage in a fast fluidized bed using X-rays and found a dilute core with a dense region on the wall. To quantitatively describe the lateral voidage profiles, Tung et al. (Tung et al., 1988) carried out measurements in a riser, and proposed an empirical correlation. Zhang et al. (Zhang et al., 1991) conducted more extensive experiments in three different fast fluidized bed systems using fiber optic probes. They found that the lateral voidage depends solely on the average cross-sectional lateral voidage. A more comprehensive correlation was proposed, which has since been widely used. Xu et al. (Xu et al., 2004) validated this correlation and found it applicable only to the dilute phase continuous clustering flow. To obtain a suitable correlation

**Table 2**  
Correlations for the lateral voidage in circulating fluidized bed systems.

Authors	Correlations	Specifications
Tung et al., 1988	$\begin{cases} \bar{\epsilon}_g = \bar{\epsilon}_g^{(r/R)^2+0.191}, & r/R \leq 0.75 \\ \bar{\epsilon}_g = \bar{\epsilon}_g^{3.62(r/R)^{6.47}+0.191}, & r/R \geq 0.75 \end{cases}$	$R$ : radius of the riser
Zhang et al., 1991	$\bar{\epsilon}_g = \bar{\epsilon}_g^{3(r/R)^{11}+(r/R)^{2.5}+0.191}$	
Patience and Chaouki, 1996	$\bar{\epsilon}_g = -4 \left( \bar{\epsilon}_g^{0.4} - \bar{\epsilon}_g \right) (r/R)^6 + \bar{\epsilon}_g^{0.4}$	
Wei et al., 1998	$\frac{1 - \bar{\epsilon}_g}{1 - \bar{\epsilon}_g} = 2.2 \frac{2}{1 + \exp(10 \cdot r/R - 7.665)}, \quad 0.68 < \bar{\epsilon}_g < 0.95$	
Xu et al., 2004	$\begin{cases} \bar{\epsilon}_g = \bar{\epsilon}_w (\epsilon_0 / \bar{\epsilon}_w)^{F(r)}, \quad \bar{\epsilon}_g < 0.75 \\ F(r) = 1.0 - 0.06(r/R) - 1.34(r/R)^2 + 2.65(r/R)^3 - 2.25(r/R)^4 \\ \bar{\epsilon}_g = \bar{\epsilon}_g^{a(r/R)^{11}+(r/R)^{2.5}+b}, \quad a = \ln \bar{\epsilon}_w / \ln \bar{\epsilon}_g - 1 - b, \quad b = \ln \epsilon_0 / \ln \bar{\epsilon}_g, \quad \bar{\epsilon}_g > 0.75 \end{cases}$	$\bar{\epsilon}_w$ : local voidage on the bed wall $\epsilon_0$ : local voidage at bed center
Issangya et al., 2005	$\bar{\epsilon}_g = \bar{\epsilon}_{mf} + (\bar{\epsilon}_g - \bar{\epsilon}_{mf}) \bar{\epsilon}_g^{-1.5+2.1(r/R)^{3.1}+5.0(r/R)^{6.8}}$	$\bar{\epsilon}_{mf}$ : voidage in incipient fluidization
Zhu et al., 2013	$\begin{cases} \frac{1 - \bar{\epsilon}_g}{1 - \bar{\epsilon}_g} = 1.264 - \frac{0.547}{1 + \exp(6.783 \cdot r/R - 4.525)}, & h/D_{is} \leq 4.77 \\ \frac{1 - \bar{\epsilon}_g}{1 - \bar{\epsilon}_g} = 1.456 - \frac{0.908}{1 + \exp(6.898 \cdot r/R - 4.721)}, & h/D_{is} > 4.77 \end{cases}$	$D_{is}$ : diameter of the riser's bottom section

for estimating the lateral voidage of the bubbling flow, Xu et al. (Xu et al., 2004) established a new correlation for the dense phase continuous bubbling flow. In Hensler et al.'s study (Hensler et al., 2016), the experimental data based on X-ray computed tomography were compared to the results calculated using classical correlations, revealing that the correlation proposed by Xu et al. had an evident deviation in predicting the solids concentration distribution in the central region of the riser.

It indicates that many classical lateral voidage correlations are based on experimental data from laboratory-scale risers. Furthermore, most reactors have a circular cross-section, and the riser's height-to-diameter ratio exceeds 15, making lateral voidage profiles highly influenced by the sidewall effect. However, industrial-scale CFB boilers have a square cross-section, and their geometric dimensions extend to tens of meters. In this context, the applicability of previous correlations remains uncertain and warrants further investigation.

For the particle lateral movement velocity, there are relatively few experimental and theoretical studies (Wei et al., 1998; Zhang and Arastoopour, 1995; Herbert et al., 1998; Zhu and Zhu, 2008; Pallarès and Johnsson, 2006; Zhou et al., 2024; Metzger et al., 2024). Pallarès and Johnsson (Pallarès and Johnsson, 2006) employed particle tracking techniques to investigate the mixing mechanism of fuel particles in a narrow "two-dimensional" fluidized bed (0.4 m  $\times$  2.15 m  $\times$  0.02 m). Zhou et al. (Zhou et al., 2024) developed a magnetic tracer-tracking system for a pilot-scale pressurized fluidized bed system and predicted the real-time velocity of the particles. Metzger et al. (Metzger et al., 2024) used magnetic resonance imaging to study the hydrodynamics of the particulate phase in three-dimensional draft tube spout-fluid beds. Tribedi et al. (Tribedi et al., 2023) introduced the radioactive particle tracking technique to measure the movement of Geldart group B particles in a pilot plant scale CFB riser. Lateral distributions of solid velocity parameters such as mean velocity, root-mean-square velocity, and granular temperature were obtained.

Although there have been some experimental studies on  $u_p$  in fluidized bed systems, most focus on laboratory-scale CFB risers, with limited experimental data. There is a lack of research on the quantitative modeling of  $u_p$ .

For strongly nonlinear systems such as CFB, an overall mathematical model based on flow simplification is an effective tool for research. However, most of the simplified mathematical models focus only on the axial distribution of gas–solid flow and axial backmixing of solids, neglecting the lateral gas–solid flow characteristics. Werther and Gungor et al. recognized the importance of lateral uniformity and developed a two-dimensional CFB mathematical model (Kruse et al., 1995; Schoenfelder et al., 1996; Kruse and Werther, 1995; Gungor and Eskin,

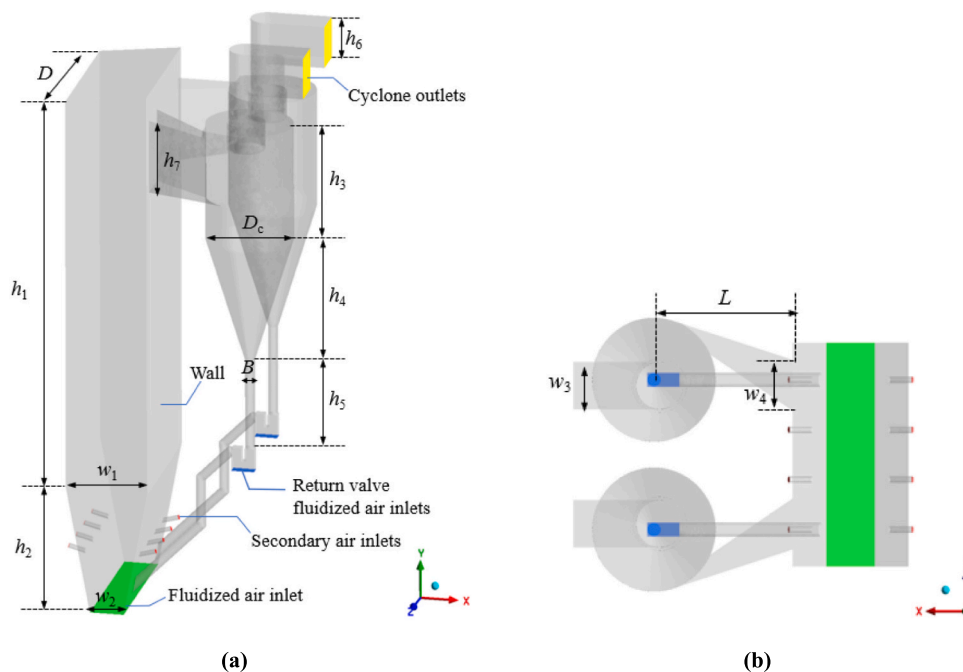


Fig. 1. Geometric model of the CFB boiler: (a) CFB boiler; (b) Top view.

Table 3  
Geometric dimensions of the CFB boiler.

Geometry	Symbol	Value (m)
Width of the entrance of the tapered segment	$w_2$	2.10
Width of the furnace	$w_1$	5.05
Depth of the furnace	$D$	9.73
Height of the tapered segment	$h_2$	7.94
Height of furnace	$h_1$	25.45
Height of the cyclone inlet	$h_7$	4.60
Width of the cyclone inlet	$w_4$	2.22
Length of the inlet duct	$L$	6.05
Diameter of the cylinder	$D_c$	5.38
Height of the cylinder	$h_3$	7.38
Height of the cone	$h_4$	7.87
Diameter of the lower outlet	$B$	0.60
Length of the lower outlet	$h_5$	5.67
Width of the cyclone outlet	$w_3$	2.06
Length of the cyclone outlet	$h_6$	2.61

2007; Gungor and Eskin, 2008). Nevertheless, in their models, the particle lateral diffusion coefficient was treated as a constant, and the lateral voidage correlations were based on limited experimental data from laboratory-scale risers. Whether these simplified approaches can be applied to the calculation of complex working conditions of CFB is questionable.

To carry out efficient and accurate parametric calculation of industrial CFB boilers, it is critical to establish a comprehensive two-dimensional model with accurate characterization of the lateral gas–solid flow. Industrial CFB boilers operate in a high-temperature and high-pressure environment, and there is a lack of effective measurement methods to obtain detailed flow field information. Given the constraints of cost, safety, and process analysis, advanced numerical experimental methods are necessary (Liu et al., 2023; Ma et al., 2023; Zhu et al., 2020). According to the multi-scale characteristics of gas–solid systems and the way of particle characterization, the simulation methods can be classified into particle-resolved direct numerical simulation (PR-DNS), Eulerian-Eulerian and Eulerian-Lagrangian methods (Shaffer et al., 2013). As a representative of the Euler-Lagrangian method, the

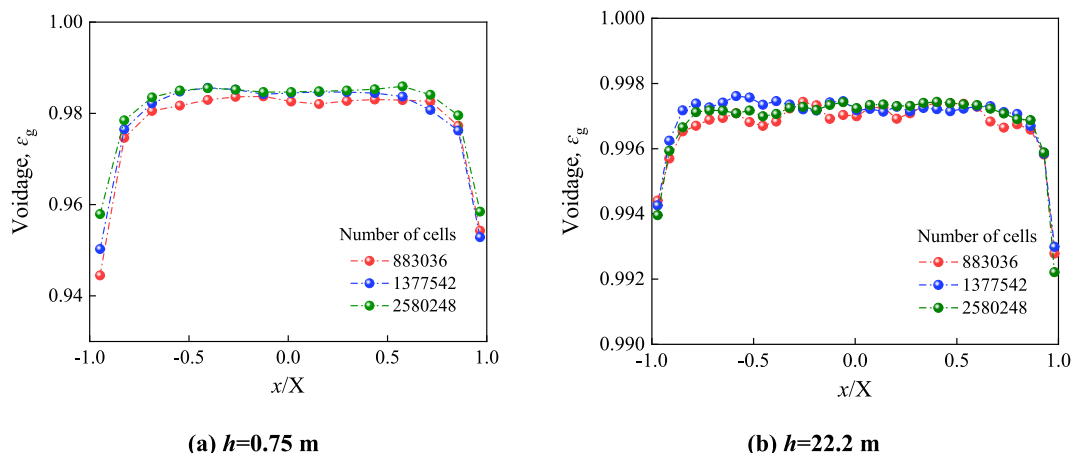


Fig. 2. Results of the grid independence test.

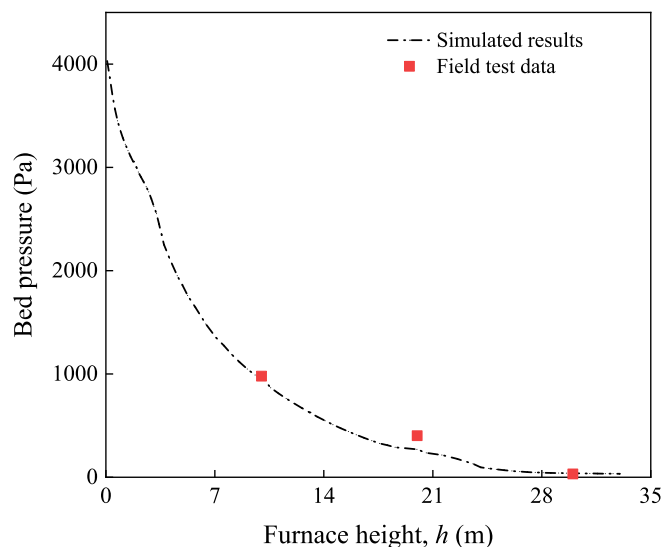
**Table 4**  
Summary of model input parameters.

Item	Parameter	Value
Simulation conditions	Initial bed inventory (m)	0.75
	Superficial gas velocity, $u_g$ (m/s)	3.13; 3.40; 3.64; 3.83; 4.08; 4.31; 4.64
	Return valve fluidized air velocity (m/s)	0.35
	Secondary air mass flow rate (kg/s)	1.964; 2.360; 2.742; 3.070; 3.550; 3.905; 4.438
	Operating temperature (K)	1165.43
	Operating pressure (Pa)	102,371
	Outlet pressure (Pa)	99,950
Gas-solid physical property parameters	Gas density ( $\text{kg}/\text{m}^3$ )	0.3141
	Gas dynamic viscosity (Pa-s)	$4.6508 \times 10^{-5}$
	Particle density ( $\text{kg}/\text{m}^3$ )	2450
	Close pack volume fraction	0.6
	Maximum momentum redirection from collision (%)	40
Particle-to-particle interaction	Particle normal stress model: Ps (Pa)	1
	Particle normal stress model: $\beta$	3
	Normal-to-wall momentum retention	0.3
	Tangent-to-wall momentum retention	0.99
Particle-to-wall interaction	Maximum iterations of volume	10
	Residual of volume	$1 \times 10^{-7}$
Solver settings	Maximum iterations of pressure	2000
	Residual of pressure	$1 \times 10^{-8}$
	Maximum iterations of velocity	50
	Residual of velocity	$1 \times 10^{-7}$
	Initial time step	$5 \times 10^{-4}$

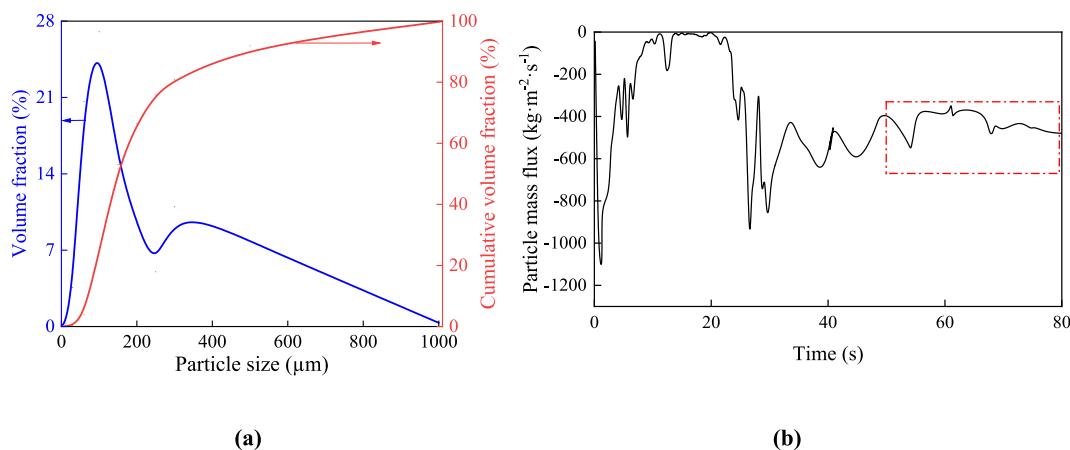
computational particle fluid dynamics (CPFD) method uses the volume-averaged Navier-Stokes equations to describe the gas phase flow and tracks the motion of particle clouds as parcels. This method can simulate gas–solid flows in industrial-scale particle systems by incorporating solid phase stress and the concept of computational particles (Liu et al., 2023). In recent years, the feasibility and accuracy of this method in simulating gas–solid flow and combustion in fluidized beds have been demonstrated (Lee et al., 2022; Klimanek and Bigda, 2018; Yang et al., 2021; Huang et al., 2025). Bai et al. (Bai et al., 2023) developed a novel solar biomass method with the solid particle heat carrier, and the gasification in fluidized bed reactor was numerically studied with CPFD model, which indicated the enhancement of internal heat transfer and reaction kinetics.

An artificial neural network (ANN) is a computational model that simulates the operational characteristics of biological neural systems. In recent decades, ANN has been widely applied in many fields such as power and energy engineering, chemical engineering, and mechanical engineering due to its remarkable advantages in identifying complex nonlinear relationships between inputs and outputs (Singh et al., 2019; Valente et al., 2014; Sedighi et al., 2017). Genetic algorithm-back propagation (GA-BP) neural network combines the advantages of genetic algorithm and back propagation neural network with high prediction accuracy and efficiency, and is widely used in data prediction, pattern recognition, and other applications (Paliwal and Kumar, 2009; Chen et al., 2020; Ni and Yang, 2021).

In this study, the CPFD method verified with field test data was employed to perform a comprehensive numerical experiment on a 170 t/h CFB boiler. The lateral voidage and particle lateral movement velocity distributions in the furnace under different loads were obtained. Based on the original simulation data set from the CPFD method, simple mathematical models for describing the lateral distributions of  $\epsilon_g$  and  $u_p$  were proposed. In addition, the GA-BP neural network was also applied to predict the  $u_p$  at different lateral positions, which is compared with the performance of theoretical model.



**Fig. 4.** Comparison of simulated results ( $u_g = 4.08$  m/s) with field test data.



**Fig. 3.** (a) Particle size distribution in calculation; (b) Particle mass flux at the lower outlet of the cyclone separator ( $u_g = 4.08$  m/s).

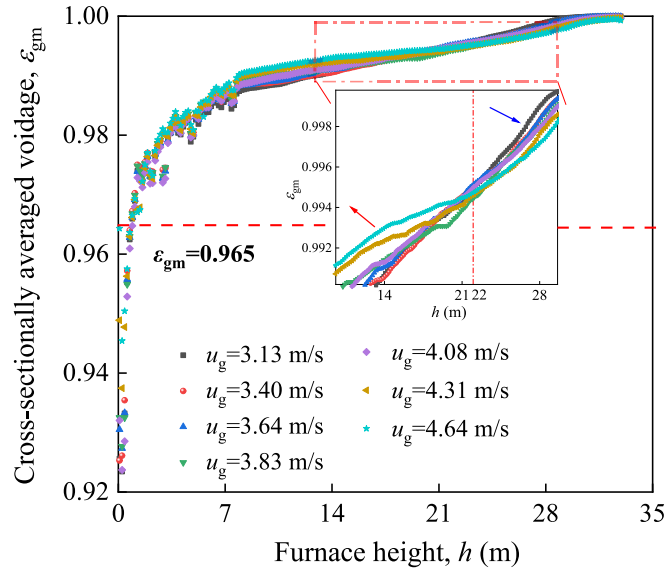


Fig. 5. Distribution of  $\epsilon_{gm}$  along the furnace height at different superficial gas velocities.

## 2. CPFD simulation

### 2.1. Governing equations

The CPFD method was proposed by Dr. Dale Snider (Snider et al., 1998; Snider, 2001) based on the multiphase particle-in-cell numerical method (Andrews and O'Rourke, 1996). In this method, the gas phase is described as a continuum under the Eulerian framework. The turbulence model applied in the present work is the large-eddy simulation (LES). The continuity equation and momentum equations for the gas phase are shown as follows:

$$\frac{\partial(\rho_g \epsilon_g)}{\partial t} + \nabla \cdot (\rho_g \epsilon_g u_g) = 0 \quad (1)$$

$$\frac{\partial(\rho_g \epsilon_g u_g)}{\partial t} + \nabla \cdot (\rho_g \epsilon_g u_g u_g) = -\nabla P + \rho_g \epsilon_g g + \nabla \cdot (\epsilon_g \tau_g) + F \quad (2)$$

where  $t$  represents time;  $\epsilon_g$ ,  $\rho_g$ ,  $u_g$ ,  $P$ ,  $\tau_g$  are the voidage, density, velocity, pressure, and stress tensor of the gas phase respectively;  $g$  is the gravitational acceleration;  $F$  represents the phase interaction force per unit volume, which can be calculated by Eq. (3):

$$F = \iiint f_p V_p \rho_p \left[ D(u_g - u_p) - \frac{\nabla P}{\rho_p} \right] dV_p d\rho_p du_p \quad (3)$$

where  $f_p$  is the particle distribution function;  $V_p$ ,  $\rho_p$ , and  $u_p$  are the volume, density, and velocity of the particle phase respectively;  $D$  is the interphase drag function.

The particle phase is treated as the discrete phase. The trajectory of the particle is calculated using Newton's equation of motion. The particle acceleration  $A_p$  is shown below:

$$A_p = \frac{du_p}{dt} = D(u_g - u_p) - \frac{\nabla P}{\rho_p} + g - \frac{\nabla \tau_p}{\epsilon_p \rho_p} \quad (4)$$

where  $\epsilon_p$  is the particle volume fraction,  $\epsilon_p = 1 - \epsilon_g$ ;  $\tau_p$  is the particle normal stress. The model of  $\tau_p$  in this work is the Harris & Crighton model (Harris and Crighton, 1994) as Eq. (5):

$$\tau_p = \frac{P_s \epsilon_p^\beta}{\max[(\epsilon_{cp} - \epsilon_p), \zeta(1 - \epsilon_p)]} \quad (5)$$

where  $P_s$  is a positive constant,  $\beta$  is a constant with a recommended value between 2 and 5.  $\zeta$  is a very small number to eliminate the

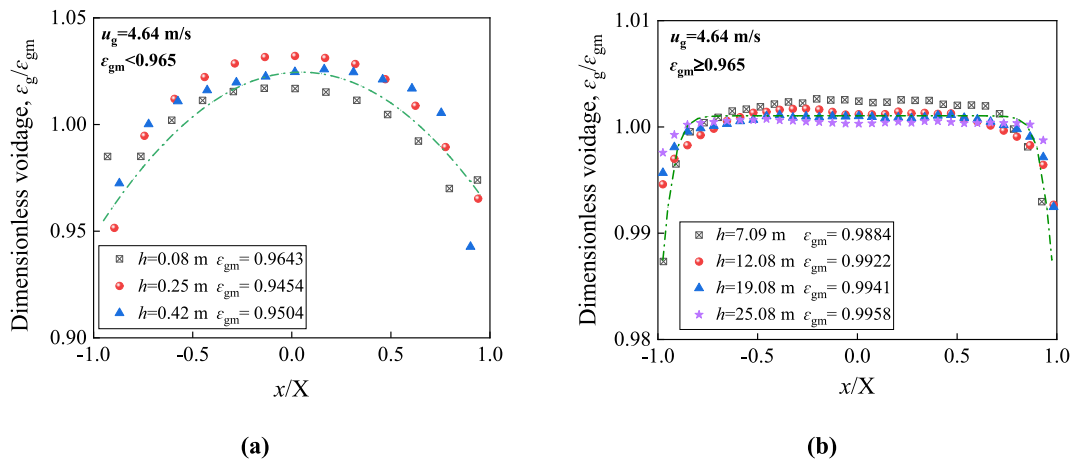
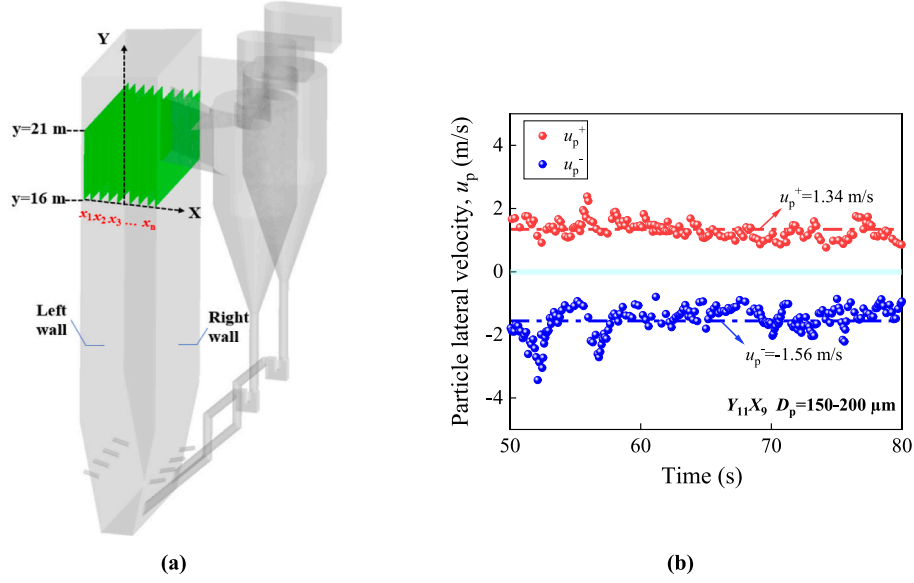


Fig. 6. Lateral profiles of the dimensionless voidage on the cross-sections: (a) Distribution of  $\epsilon_g/\epsilon_{gm}$  in the dense phase region; (b) Distribution of  $\epsilon_g/\epsilon_{gm}$  in the dilute phase region.



**Fig. 7.** Distribution of particle lateral velocity on the monitoring surface  $Y_{11}X_9$ : (a) Monitoring surface settings; (b) Lateral velocity of particles with a size-cut group of 150–200  $\mu\text{m}$ .

computation singularity.

The particle position  $x_p$  is calculated by Eq. (6):

$$\frac{dx_p}{dt} = u_p$$

$$C_d = \begin{cases} \frac{24}{\text{Re}_p} \varepsilon_g^{-2.65}, & \text{Re}_p < 0.5 \\ \frac{24}{\text{Re}_p} (1 + 0.15\text{Re}_p^{0.687}) \varepsilon_g^{-2.65}, & 0.5 \leq \text{Re}_p \leq 1000 \\ 0.44 \varepsilon_g^{-2.65}, & \text{Re}_p > 1000 \end{cases} \quad (8)$$

## 2.2. Drag model

Drag force is the primary interaction force between the gas and solid phases. Calculating the drag force is crucial for determining the accuracy of the simulation. Various drag models have been developed, including Stokes (White, 1991), Wen-Yu (Wen and Yu, 1966), Ergun (Beetstra et al., 2007), and the Wen-Yu/Ergun hybrid model (Gidaspow, 1994). Among them, the Wen-Yu/Ergun mixed model applies to a wide range of particle concentrations, and its accuracy in predicting the gas–solid flow behavior in CFB has been verified in recent years (Ma et al., 2018; Yan et al., 2020). Therefore, the Wen-Yu/Ergun hybrid model was chosen in this study to calculate the drag force, as shown below:

where  $\text{Re}_p$  represents the particle Reynolds number, which can be calculated by Eq. (9):

$$\text{Re}_p = \frac{2\rho_g r_p |u_g - u_p|}{\mu_g} \quad (9)$$

where  $\mu_g$  is the dynamic viscosity of the gas phase.

## 2.3. Geometry and mesh

The simulation was performed using the software Barracuda®

$$D = \begin{cases} D_{\text{Wen-Yu}} = \frac{3}{8} C_d \frac{\rho_g}{\rho_p} \frac{|u_g - u_p|}{r_p}, & \varepsilon_p < 0.75\varepsilon_{cp} \\ \frac{\varepsilon_p - 0.75\varepsilon_{cp}}{0.85\varepsilon_{cp} - 0.75\varepsilon_{cp}} (D_{\text{Ergun}} - D_{\text{Wen-Yu}}) + D_{\text{Wen-Yu}}, & 0.75\varepsilon_{cp} < \varepsilon_p < 0.85\varepsilon_{cp} \\ D_{\text{Ergun}} = 0.5 \left[ \frac{180\varepsilon_p}{(1 - \varepsilon_p)\text{Re}_p} + 2 \right] \frac{\rho_g}{\rho_p} \frac{|u_g - u_p|}{r_p}, & \varepsilon_p > 0.85\varepsilon_{cp} \end{cases} \quad (7)$$

where  $D_{\text{Wen-Yu}}$  is the drag function in the Wen-Yu drag model;  $D_{\text{Ergun}}$  is the drag function in the Ergun drag model;  $r_p$  is the particle radius;  $C_d$  is the drag coefficient, which can be calculated according to the Eq. (8):

17.4.0. A 170t/h CFB boiler was modeled in a 1:1 ratio. The geometric model is shown in Fig. 1, and the main geometric parameters of the boiler are listed in Table 3.

The built-in grid division function of Barracuda was used to generate three different sizes of mesh in the Cartesian coordinate system: coarse mesh, medium mesh, and fine mesh. The corresponding effective mesh numbers were 883,036, 1,377,542, and 2,580,248, respectively. The lateral profiles of the voidage  $\varepsilon_g$  on the cross-sections with furnace heights of 0.75 m and 22.2 m under different meshes are shown in Fig. 2

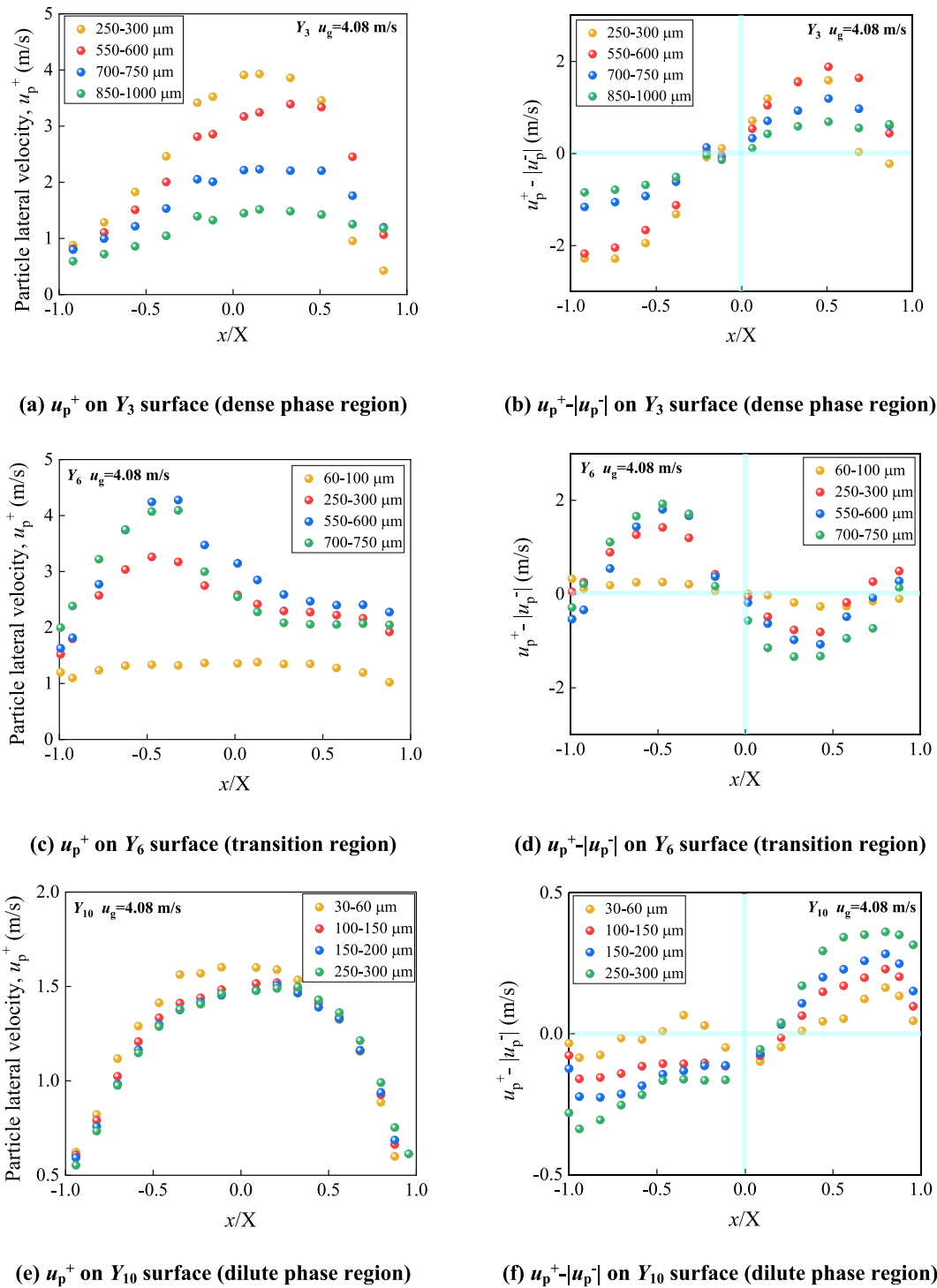


Fig. 8. Particle lateral velocity distribution in different regions: (a, b) for  $Y_3$  surface in the dense phase region; (c, d) for  $Y_6$  surface in the transition region; (e, f) for  $Y_{10}$  surface in the dilute phase region.

(a) and Fig. 2b respectively. It can be seen that the calculation results of the medium mesh are close to those of the fine mesh. Considering both the calculation accuracy and calculation cost, the medium mesh was chosen for the present study.

#### 2.4. Boundary conditions

Fig. 1(a) demonstrates the setting of boundary conditions for the computational model. In the initial state, a certain height of solids is piled up at the bottom of the furnace. The air enters from the bottom of

the furnace, the return valve, and the secondary air ports at specific velocities, respectively. A small number of fine particles escape through the separator outlet. Both fluidized air inlet and return valve fluidized air inlets are set as velocity inlets with specific velocities, as shown in Table 4; to make the calculation more stable, the secondary air inlets are set as mass flow inlets with specific mass flow rates; the cyclone outlets are set as pressure outlets with a fluid pressure of 99950 Pa. The particle size distribution in the calculation is shown in Fig. 3(a). The physical properties of the particles and gas, along with simulation conditions, solver settings, and other important input parameters are summarized in

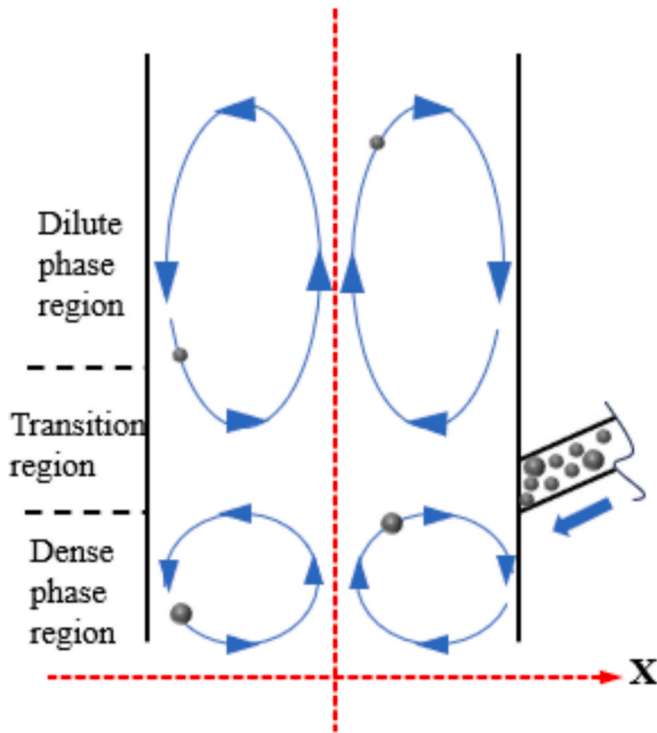


Fig. 9. Particle internal flow structures in different regions.

Table 4.

The total simulation time was 80 s. The initial time step was  $5 \times 10^{-4}$  s, and the Courant number CFL was set to 0.8–1.5. The particle mass flux at the lower outlet of the cyclone separator was monitored. As shown in Fig. 3(b), with the increase in the calculation time, the particle mass flux gradually reached stability after 45 s. Therefore, the average data from 50 s to 80 s were extracted as the calculation results under quasi-steady state.

### 2.5. Model validation

Before conducting the formal calculations, the measured bed pressure from the same industrial boiler was used to validate the CPF model. Fig. 4 shows the comparison between the simulated values and the field test data. The CPF model-calculated bed pressure distribution is consistent with the measured data, verifying the reliability of the numerical model to some extent.

## 3. Characterization of lateral voidage distribution

### 3.1. Numerically calculated distributions

Numerous studies have shown that the lateral voidage profiles depend solely on the cross-sectionally averaged voidage  $\epsilon_{gm}$  exclusively, and are independent of the operating conditions, solid properties, and fluidized bed diameter (Tung et al., 1988; Zhang et al., 1991; Xu et al., 2004). Fig. 5 exhibits the distribution of the cross-sectionally averaged voidage  $\epsilon_{gm}$  along the furnace height at different superficial gas velocities. In the dense phase region,  $\epsilon_{gm}$  is nearly constant, while in the dilute phase region, it exhibits an exponential growth with increasing furnace height. This distribution feature is consistent with many experimental results (Wong et al., 1992; Xu et al., 2018). With the increase of superficial gas velocity,  $\epsilon_{gm}$  increases in the middle and lower regions of the furnace ( $h < 22$  m), while it decreases in the upper region of the furnace.

The lateral profiles of the dimensionless voidage  $\epsilon_g/\epsilon_{gm}$  on some

specific cross-sections are presented in Fig. 6. Overall, the lateral profiles of  $\epsilon_g/\epsilon_{gm}$  exhibit distinct distribution characteristics in the dense phase continuous bubbling flow region and the dilute phase continuous clustering flow region. In the dense phase region, the lateral voidage distribution follows the form of a parabola, consistent with previous research (Hartge et al., 1988; Tung et al., 1988; Zhang et al., 1991; Xu et al., 2004; Hensler et al., 2016). Different from it, in the dilute-phase region, although the lateral voidage remains high in the sidewall region, the distribution in the center is nearly flat, and a sharp transition between the sidewall and center regions occurs, which is different from the situation in previous studies (Zhang et al., 1991; Wei et al., 1998; Patience and Chaouki, 1996; Wang et al., 1998).

### 3.2. Empirical correlation

To quantitatively describe the lateral voidage profiles in a CFB, two distinct hydrodynamic regions, delineated at  $\epsilon_{gm} = 0.965$ , are identified based on the characteristic variations in the profiles. When  $\epsilon_{gm} < 0.965$ , the dense phase continuous bubbling flow prevails, which can be described by a polynomial. In contrast, when  $\epsilon_{gm} \geq 0.965$ , the voidage in the center region remains relatively constant, while the voidage in the side wall region decreases rapidly. Given this distribution feature, the Boltzmann distribution function is used for modeling. Based on the above analysis, the correlations for lateral voidage profiles are established as follows:

$$\epsilon_{gm} < 0.965 \frac{1 - \epsilon_g}{1 - \epsilon_{gm}} = \begin{cases} a \left(\frac{x}{X}\right)^2 + b \left(\frac{x}{X}\right) + c, & -1 \leq \frac{x}{X} \leq 1 \\ a = 0.611 + e^{1.752Re_p - 8.425} \\ b = -0.05Re_p^3 + 0.50Re_p^2 - 1.577Re_p + 1.570 \\ c = 0.402 + \frac{0.267}{1 + e^{6.882Re_p - 30.278}} \end{cases} \quad (10)$$

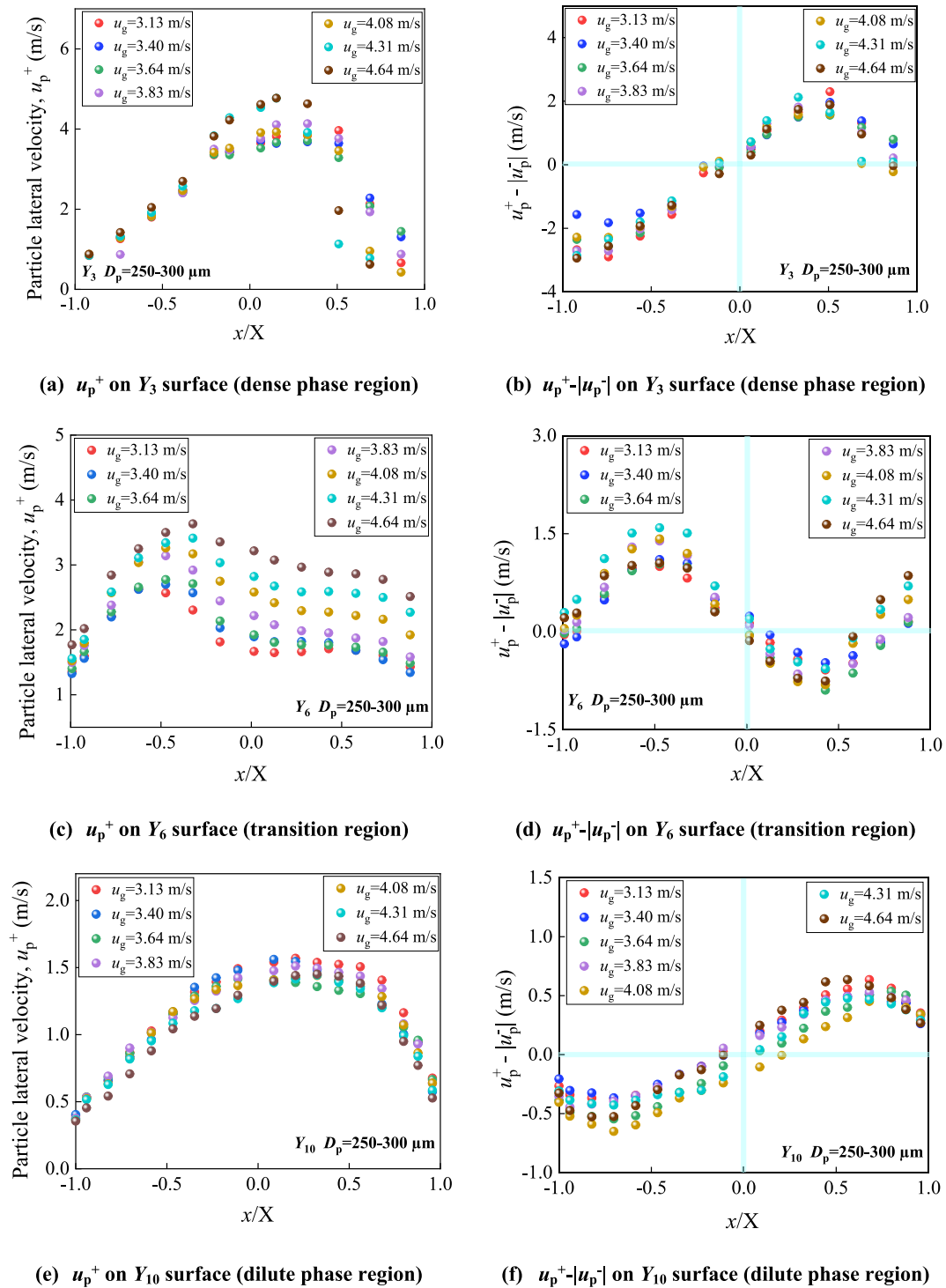
$$\epsilon_{gm} \geq 0.965 \frac{1 - \epsilon_g}{1 - \epsilon_{gm}} = \begin{cases} 0.858 + \frac{5.651}{1 + e^{24.713\left(\frac{x}{X}\right) + 24.830}}, & -1 \leq \frac{x}{X} \leq 0 \\ 0.858 + \frac{5.651}{1 + e^{-24.713\left(\frac{x}{X}\right) + 24.830}}, & 0 < \frac{x}{X} \leq 1 \end{cases} \quad (11)$$

## 4. Characterization of particle lateral movement velocity

### 4.1. Numerically calculated distributions

To obtain the particle lateral velocity in the furnace, several monitoring surfaces were set up along the lateral direction at different furnace heights, denoted as  $Y_1, Y_2, \dots, Y_n$ , as shown in Fig. 7(a) (taking  $Y_{11}$  ( $16 \text{ m} \leq h \leq 21 \text{ m}$ ) as an example). For the lateral direction, monitoring surfaces are set up at different  $x$ -coordinate positions, denoted as  $Y_{11}X_1, Y_{11}X_2, \dots, Y_{11}X_n$ . Following this approach, a total of 210 monitoring surfaces were set up along the furnace height and in the lateral direction. The coordinates of all monitoring surfaces are detailed in Table S1, Supplementary Material (SM). Particle information (unique particle ID, diameter, velocity vector, residence time, etc.) passing through the monitoring surfaces at different times is recorded, accordingly, the  $u_p$  distributions at various times and radii can be obtained. For convenience, the particles are divided into 19 size-cut groups according to the diameter, from 0 to 1050  $\mu\text{m}$ .

Fig. 7(b) shows the  $u_p$  with a size-cut group of 150–200  $\mu\text{m}$  passing through the surface  $Y_{11}X_9$  within the time interval of 50–80 s. It can be seen that during this period, the  $u_p$  fluctuates, but most values are within a narrow range. Hence, in this paper, the average value of  $u_p$  in this period is used for the following analysis. Note that positive values



**Fig. 10.** Particle lateral velocity distribution in different regions under different superficial gas velocities: (a, b) for  $Y_3$  surface in the dense phase region; (c, d) for  $Y_6$  surface in the transition region; (e, f) for  $Y_{10}$  surface in the dilute phase region.

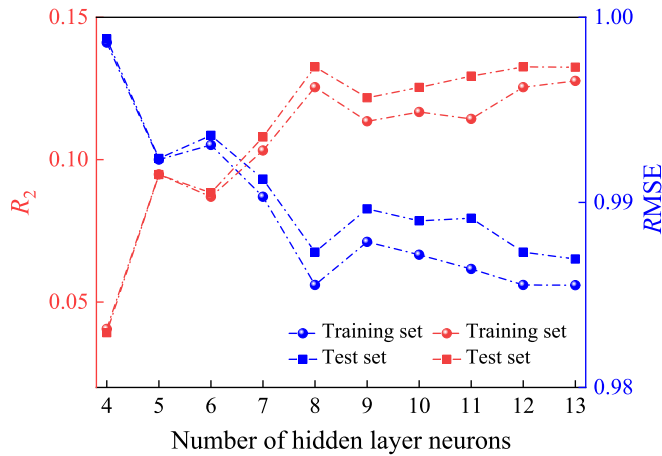
indicate particles move in the positive direction of the X-axis.

Based on different distribution characteristics, the furnace can be divided into the dense phase region, the transition region, and the dilute phase region. Fig. 8 shows  $u_p$  distributions for particles with different sizes on some representative monitoring surfaces in different regions. Meanwhile, the velocity difference at the same lateral position ( $u_p^+ - |u_p^-|$ ) is also given.  $x/X = 0$  represents the position of the vertical central surface inside the furnace and  $x/X > 0$  points to the area near the right wall.

As shown in Fig. 8(a), in the dense phase region,  $u_p$  profiles for different size particles are roughly in the parabolic form. Fig. 8(b) further shows that, when  $r/R > 0$ ,  $u_p^+ > u_p^-$ ; when  $r/R < 0$ ,  $u_p^+ < u_p^-$ . Namely, more particles tend to move laterally towards the wall, which leads to an increase in the particle concentration near the wall, and subsequently forms an internal solids circulation in the bottom bed, as shown in Fig. 9. It is observed that the distribution curves in Fig. 8(b) are not strictly centrally symmetric. This may be related to the fact that the particle return port is arranged on the right wall, that is, a large number

**Table 5**  
Particle lateral movement velocity calculation model.

Region	Expression
$0.920 \leq \varepsilon_{gm} < 0.965$	$-u_g \ln(u_g - u_p^+) - u_p^+ = \left( \frac{8.091 \varepsilon_g^{-2.65}}{r_p^2} \times 10^{-12} - \frac{0.00281 \varepsilon_g^{-2.65}  u_g - u_p^+ ^{0.687}}{r_p} \right) x - 0.96678$ $u_g \ln(u_g - u_p^-) + u_p^- = \left( \frac{1.514 \varepsilon_g^{-2.65}}{r_p^2} \times 10^{-11} - \frac{0.0038484 \varepsilon_g^{-2.65}  u_g - u_p^- ^{0.687}}{r_p} \right) x - 0.82914$
$0.965 \leq \varepsilon_{gm} < 0.990$	$-u_g \ln(u_g - u_p^+) - u_p^+ = \left( \frac{2.157 \varepsilon_g^{-2.65}}{r_p^2} \times 10^{-12} + \frac{0.000123 \varepsilon_g^{-2.65}  u_g - u_p^+ ^{0.687}}{r_p} \right) x - 1.61841$ $u_g \ln(u_g - u_p^-) + u_p^- = \left( \frac{8.541 \varepsilon_g^{-2.65}}{r_p^2} \times 10^{-12} - \frac{0.001524 \varepsilon_g^{-2.65}  u_g - u_p^- ^{0.687}}{r_p} \right) x - 1.54211$
$\varepsilon_{gm} \geq 0.990$	$-u_g \ln(u_g - u_p^+) - u_p^+ = \left( \frac{8.116 \varepsilon_g^{-2.65}}{r_p^2} \times 10^{-12} + \frac{0.00267 \varepsilon_g^{-2.65}  u_g - u_p^+ ^{0.687}}{r_p} \right) x - 0.65773$ $u_g \ln(u_g - u_p^-) + u_p^- = \left( -\frac{8.685 \varepsilon_g^{-2.65}}{r_p^2} \times 10^{-12} + \frac{0.002405 \varepsilon_g^{-2.65}  u_g - u_p^- ^{0.687}}{r_p} \right) x - 0.7882$



**Fig. 11.** RMSE and  $R_2$  values of the test set and training set at different numbers of hidden layer neurons.

**Table 6**  
Summary of the GA-BP neural network parameters.

Parameter	Description
ANN type	Single hidden layer feed forward neural network
Learning type	Supervised learning
Training algorithm	Levenberg-Marquardt
Number of input layer neurons	5
Number of hidden layer neurons	13
Number of output layer neurons	1
Activation function for the hidden layer	Logsig function
Activation function for the output layer	Tansig function
Maximum number of training epochs	20,000
Learning rate	0.0005
Population size	50
Maximum number of generations	50
Parameter of the selection function	0.09
Parameter of the cross function	2
Parameter of the variance function	[2 gen 3]

of particles rush into the furnace laterally from the right side, causing particles in the dense phase zone to tend to flow to the left.

Similar to the particle flow pattern in the dense phase region, a core-annulus flow pattern is formed in the dilute phase region. Namely, the particle flow rate towards the wall is greater than that towards the furnace center, resulting in the formation of a downward flow of dense-phase clusters near the wall. This phenomenon has also been confirmed by numerous experiments (Bai et al., 1995; Kim et al., 2004; Breault et al., 2020).

Different from the phenomena shown in the above two regions, in the transition region, the lateral movement direction of particles is opposite, demonstrating an obvious centripetal particle flow. This is because, under the influence of factors such as the central upward airflow and the secondary air injection, the particle clusters falling from the side walls are broken up and move inward and upward, until they enter the dilute phase region and re-agglomerate, thus forming a complete solid in-furnace circulation, as shown in Fig. 9.

Particle size and superficial gas velocity are key factors that control particle flow characteristics. As shown in Fig. 8, particles with different sizes exhibit varying velocity distributions. In the dilute phase region, the lateral velocity of fine particles is higher than that of coarse particles. Fine particles have a lower terminal velocity, making them more easily carried by the airflow. In the dense phase and transition regions, the situation is somewhat different. Due to the high particle concentration, the effect of inter-particle collisions on lateral velocity becomes significant. Coarse particles are more susceptible to collisions and thus become more active, resulting in a significantly higher lateral velocity compared to the dilute-phase region.

In this study, different boiler loads (boiler load ratio changes from 50 % to 110 %) are considered in the simulation, thus discussing the effects of superficial gas velocity on the fluid dynamics. Fig. 10 shows that, in the dense phase and transition regions, the higher the superficial gas velocity, the greater the gas's ability to carry particles, resulting in a general increase in  $u_p$ . Nevertheless, in the dilute phase region, at high superficial gas velocities, such as  $u_g = 4.64$  m/s,  $u_p$  is slightly lower than that at relatively lower superficial gas velocity.

Other than particle size and gas velocity, some other factors, such as solid suspension concentration, also have significant impacts on the  $u_p$  distribution (Parssinen and Zhu, 2001). It can be seen that there are numerous interrelated influencing factors for  $u_p$ . Different from the description of voidage profiles, it is very difficult to accurately estimate the  $u_p$  value at any position in the furnace under various conditions using a unified correlation equation. Therefore, this paper attempts two technical approaches respectively, namely physical equation derivation

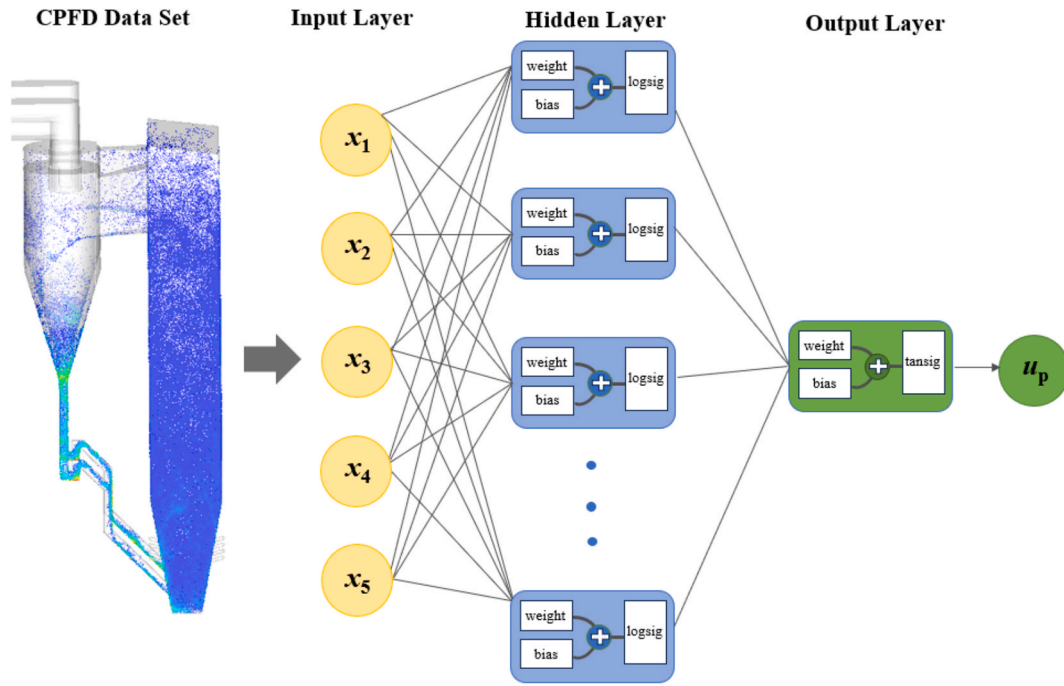


Fig. 12. Schematic diagram of the constructed GA-BP neural network.

and machine learning, to obtain a  $u_p$  prediction model that applies to a wide range of working conditions and the whole domain.

#### 4.2. Theoretical calculation model

For the gas–solid two-phase flow, the drag force mainly drives the particle lateral movement, and the Newtonian motion equation of the particles is expressed as follows:

$$\frac{du_p}{dt} = D(u_g - u_p) \quad (12)$$

It can be further converted into Eq. (13):

$$\frac{du_p}{dx} \cdot u_p = D(u_g - u_p) \quad (13)$$

$Re_p$  in a CFB boiler furnace is mainly in the range of 0.5–1000, so the corresponding drag function  $D$  can be written as follows (Gidaspow, 1994):

$$D = \frac{9 \mu_g \varepsilon_g^{-2.65}}{2 \rho_p r_p^2} + \frac{27 \rho_g \varepsilon_g^{-2.65} |u_g - u_p|^{0.687}}{20 \rho_p r_p} \quad (14)$$

Substituting Eq. (14) into Eq. (13) gives the following equation:

$$\frac{u_p}{u_g - u_p} du_p = D dx = \left( \frac{9 \mu_g \varepsilon_g^{-2.65}}{2 \rho_p r_p^2} + \frac{27 \rho_g \varepsilon_g^{-2.65} |u_g - u_p|^{0.687}}{20 \rho_p r_p} \right) dx \quad (15)$$

Eq. (15) is a first-order linear non-homogeneous differential equation. For the right side of the equation, considering that the second term contains the absolute value of the variable  $u_p$ , no analytical solution can be obtained if it is solved directly. Therefore, in this paper, the whole term  $|u_g - u_p|^{0.687}$  is regarded as a new variable  $\alpha$  for simplification, and the differential equation is solved as follows:

$$-u_g \ln(u_g - u_p) - u_p = \left( \frac{9 \mu_g \varepsilon_g^{-2.65}}{2 \rho_p r_p^2} + \frac{27 \rho_g \varepsilon_g^{-2.65} \alpha}{20 \rho_p r_p} \right) x + C \quad (16)$$

This equation is an implicit function of  $u_p$ , in which the gas velocity, voidage, particle lateral position,  $\alpha$ , and particle radius can be regarded as independent variables  $x_1$ ,  $x_2$ ,  $x_3$ ,  $x_4$ , and  $x_5$ , respectively, as shown

below. Substituting the corresponding CPFD calculation data into this equation and solving the unknown coefficients  $A$ ,  $B$ , and  $C$ , the distribution expression of  $u_p$  can be obtained.

$$-x_1 \ln(x_1 - y) - y = \left( \frac{Ax_2^{-2.65}}{x_5^2} + \frac{Bx_2^{-2.65}x_4}{x_5} \right) x_3 + C \quad (17)$$

As discussed in Section 4.1,  $u_p$  exhibits different distribution characteristics across various regions. Therefore, for regions featured by different solid suspension densities, the coefficients in the implicit equations related to  $u_p$  are also discrepant. Table 5 provides the  $u_p$  calculation method derived from the physical model. The  $R_2$  values of all expressions exceeded 0.9, indicating that the established model has excellent accuracy.

#### 4.3. GA-BP neural network

##### 4.3.1. Model structure

The computing units in ANN are called neurons. Among them, the neurons responsible for receiving external information are defined as input layer neurons, and the neurons taking charge of the output of information are called output layer neurons (Ganesan et al., 2015). A neuron calculates the weighted sum of its inputs and then passes it to the activation function to obtain the neuron output. Mathematically, the output  $y_k$  of the  $k$ th neuron can be expressed as Eq. (18):

$$y_k = f(\beta_k) = f\left(\sum_{i=1}^d (w_{ik}x_i + b_k)\right) \quad (18)$$

where  $x_i$  are the input signals,  $w_{ik}$  are the connection weights of the neurons,  $f$  is the activation function, and  $b_k$  are the biases.

An ANN typically consists of three or more layers: the input layer, the hidden layer, and the output layer (Paliwal and Kumar, 2009). In this study, based on the relationship between each influencing factor and  $u_p$ , the input layer signals are finalized as gas velocity, voidage, particle lateral position,  $\beta$ , and particle radius, while the  $u_p$  is defined as the output layer signal. The hidden layer processes the input data; the more hidden layers there are, the more comprehensive the information that can be processed. However, selecting too many hidden layers may lead

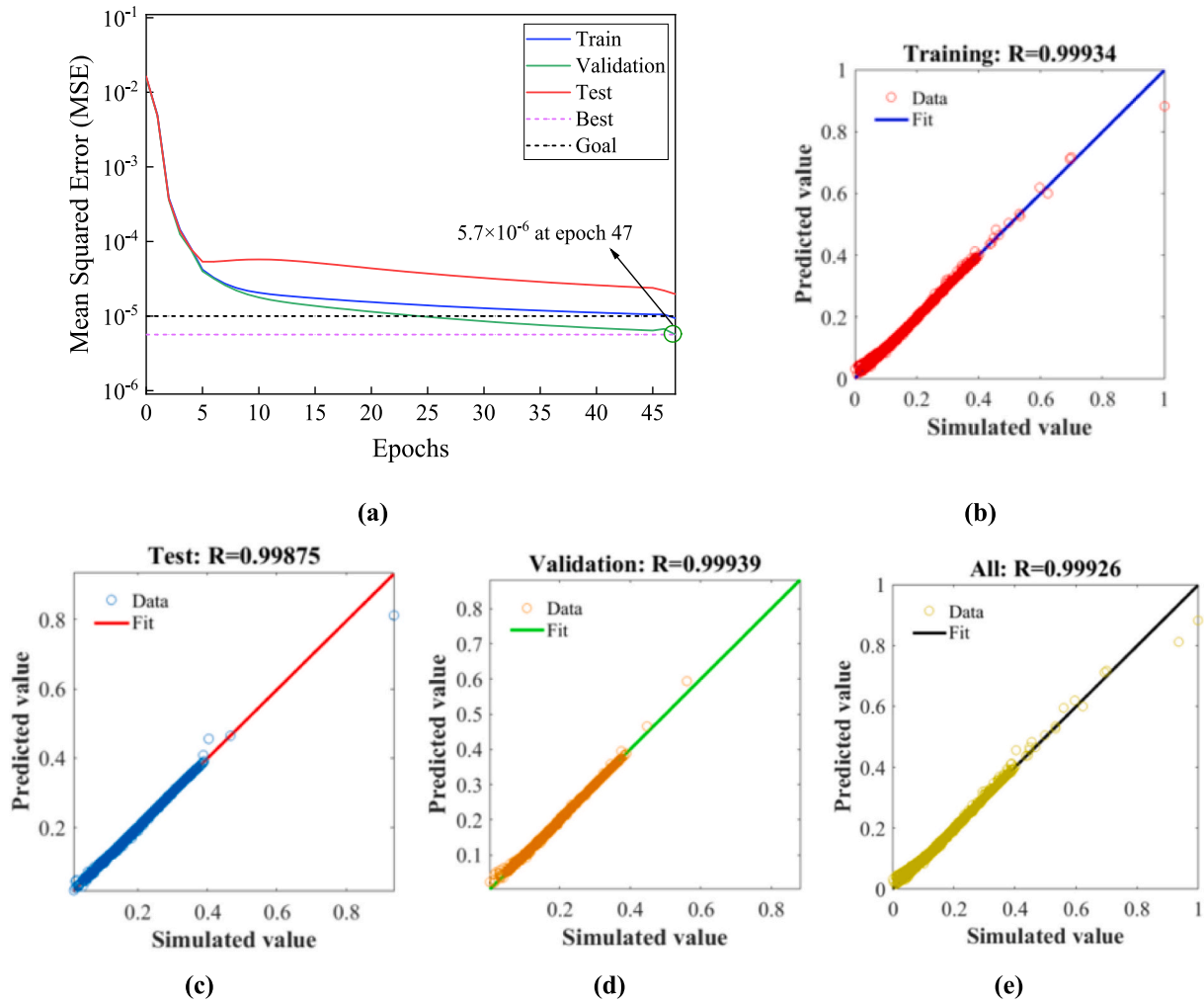


Fig. 13. Training performance of the GA-BP neural network: (a) Mean squared error (MSE) during ANN training; (b)  $R$  of the training set; (c)  $R$  of the test set; (d)  $R$  of the validation set; (e)  $R$  of all sets.

to overfitting. Generally, a neural network topology with a single hidden layer is sufficient to solve most problems. Therefore, in this study, the number of hidden layers was set to 1. For hidden layer neurons, the number  $q$  is typically determined by the following equation:

$$q = \sqrt{m+n} + a \quad (19)$$

where  $m$  and  $n$  are the number of input layer nodes and output layer nodes respectively, and  $a$  is an integer in 1–10. In this study,  $q = 4$ –13, so the number of hidden layer neurons is selected in 4–13.

Commonly used training algorithms in ANN include Levenberg-Marquardt (LM), scaled conjugate gradients (SCG), gradient descent with momentum, and adaptive learning rules (GDx) (Hamad et al., 2017; Shabanpour et al., 2017; Worden and Barton, 2004). Among these, the LM algorithm has a high calculation speed and learning accuracy and is used in this study to train the ANN.

The data set used in the model comes from the CPF simulation, of which 70 % is used for training, 15 % is used for testing, and 15 % is for validation. To eliminate the effect of consistent data sequence on model training parameters, the data are randomly sorted in each training and testing stage. Additionally, to accelerate convergence, the data are normalized using Eq. (20) before training the neural network:

$$x_{\text{norm}} = \frac{x - x_{\min}}{x_{\max} - x_{\min}} \quad (20)$$

where  $x_{\text{norm}}$ ,  $x_{\max}$ , and  $x_{\min}$  are the normalized, maximum, and

minimum values of the input respectively.

The root mean square error (RMSE) and coefficient of determination ( $R_2$ ) are used to evaluate the prediction performance of the model:

$$\text{RMSE} = \sqrt{\frac{1}{N} \sum_{i=1}^N (y_i - f_i)^2} \quad (21)$$

$$R^2 = 1 - \frac{\sum_{i=1}^N (y_i - f_i)^2}{\sum_{i=1}^N (y_i - \bar{y}_i)^2} \quad (22)$$

where  $y_i$  and  $f_i$  are the predicted and target values, respectively; and  $\bar{y}_i$  is the mean value of  $y_i$ .

Fig. 11 shows the RMSE and  $R_2$  values of  $u_p$  for both the test set and training set when the number of hidden layer neurons ranged from 4 to 13. The minimum RMSE and maximum  $R_2$  values in the test set both come from the neural network architecture with hidden layer neurons  $q = 13$ , demonstrating that the prediction performance is optimal under this network architecture. Therefore,  $q$  is set to 13 in this study.

The detailed information on various parameters used to construct the GA-BP neural network is summarized in Table 6, and the schematic diagram of the constructed GA-BP neural network is shown in Fig. 12.

#### 4.3.2. Prediction results

Based on the previous discussion in Section 4.1, the CPF data set is classified according to the distribution characteristics of  $u_p$ , and it is used

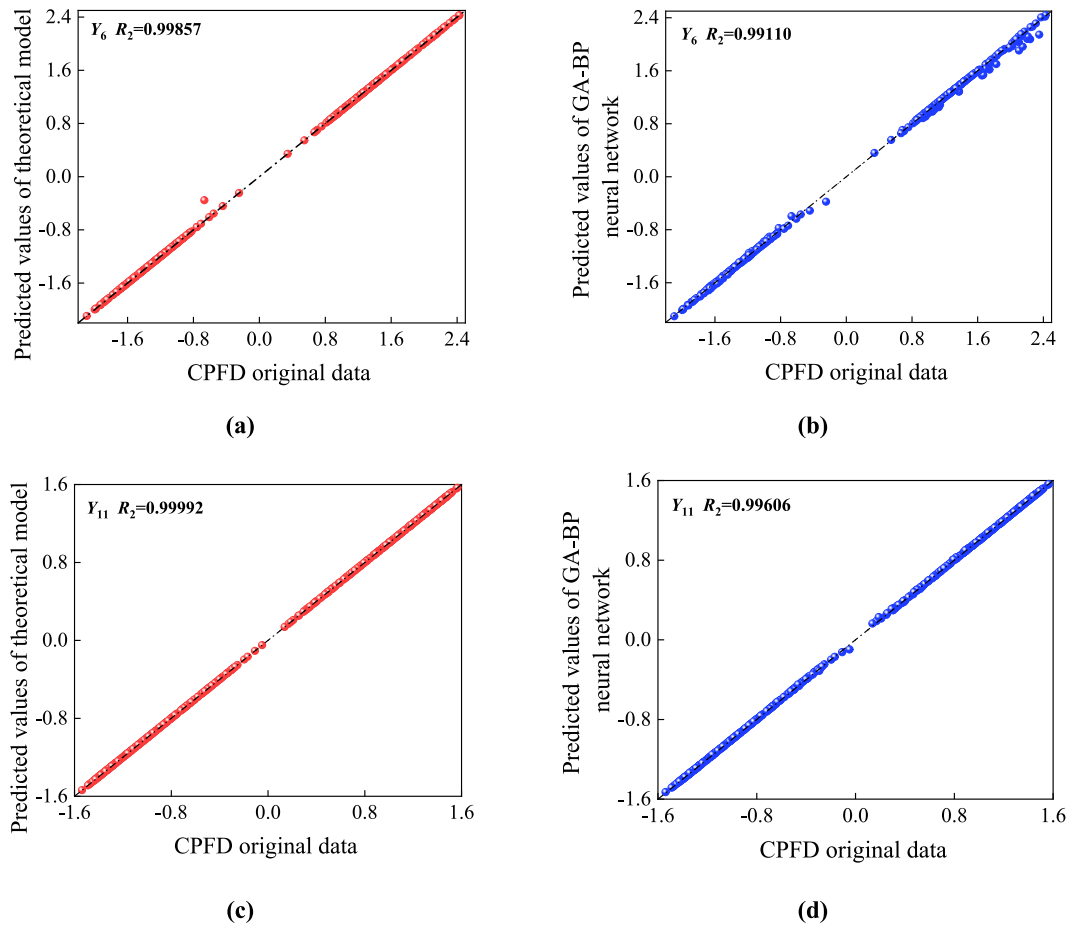


Fig. 14. Comparison of prediction performance between the theoretical model and GA-BP neural network: (a)  $u_p$  on  $Y_6$  predicted by theoretical model; (b)  $u_p$  on  $Y_6$  predicted by GA-BP neural network; (c)  $u_p$  on  $Y_{11}$  predicted by theoretical model; (d)  $u_p$  on  $Y_{11}$  predicted by GA-BP neural network.

to train the GA-BP neural network. After iteration and optimization, the connection weights, biases, and the corresponding RMSE and  $R_2$  values are obtained and listed in Table S2 (SM). Fig. 13 illustrates the training performance of the model in predicting the particle lateral velocity distribution in the dilute phase region. It shows that the model reaches a minimum error of  $5.7 \times 10^{-6}$  after 32 iterations, satisfying the convergence requirement. Furthermore, the  $R_2$  values for the training set, validation set, test set, and all sets exceed 0.99, indicating excellent prediction performance.

#### 4.4. Comparison of theoretical model and GA-BP neural network model

To quantitatively characterize the  $u_p$  distribution, two models are developed based on physical model derivation and GA-BP neural network, respectively. To select an optimal model for integrating it into the two-dimensional CFB mathematical model, both methods are evaluated in terms of accuracy and applicability.

The operating performance of the boiler at 30 % load was calculated using the verified CPFD method, and the  $u_p$  distribution was obtained. Fig. 14 shows the predicted  $u_p$  on monitoring surfaces  $Y_6$  and  $Y_{11}$  using the above two methods. Both models exhibit excellent prediction performance, with  $R_2$  exceeding 0.99. Regarding applicability, the  $u_p$  calculation method based on the physical model is relatively simple, involving five independent variables, which has a clear physical meaning and strong universality. The GA-BP neural network reversely solves the neural network using the trained weights and biases, whose calculation process is also straightforward. However, compared to the physical model, the GA-BP neural network has a strong dependence on the data set and lacks advantages in terms of generalization and

applicability to other scenarios.

Given that the theoretical calculation model based on physical functions has evident simplicity, universality, and high accuracy, it is strongly recommended to utilize this model for predicting the  $u_p$  distribution within the CFB boiler furnace. Moreover, it will be integrated into the evolving two-dimensional CFB mathematical model.

## 5. Conclusions

To accurately describe the lateral gas–solid flow characteristics in industrial CFB boilers, a 170 t/h CFB boiler was numerically simulated using the CPFD method validated by field test data. The lateral voidage profiles and particle lateral movement velocity distributions inside the furnace under wide operation conditions were thoroughly investigated, and simple models were developed to predict them.

When the cross-sectionally averaged voidage  $\varepsilon_{gm} < 0.965$ , the lateral profiles of  $\varepsilon_g$  follow a parabolic form. When  $\varepsilon_{gm} \geq 0.965$ , its distribution resembles a Boltzmann function, with a flat distribution of  $\varepsilon_g$  in the center region and a sharp decrease in the side wall region. Correlations for the lateral voidage profiles are proposed as follows:

$$\varepsilon_{gm} < 0.965 \quad \frac{1 - \varepsilon_g}{1 - \varepsilon_{gm}} = \begin{cases} a \left(\frac{x}{X}\right)^2 + b \left(\frac{x}{X}\right) + c, & -1 \leq \frac{x}{X} \leq 1 \\ a = 0.611 + e^{1.752\text{Re}_p - 8.425} \\ b = -0.05\text{Re}_p^3 + 0.50\text{Re}_p^2 - 1.577\text{Re}_p + 1.570 \\ c = 0.402 + \frac{0.267}{1 + e^{6.882\text{Re}_p - 30.278}} \end{cases}$$

$$\varepsilon_{gm} \geq 0.965 \frac{1 - \varepsilon_g}{1 - \varepsilon_{gm}} = \begin{cases} 0.858 + \frac{5.651}{1 + e^{24.713\left(\frac{x}{X}\right) + 24.830}}, & -1 \leq \frac{x}{X} \leq 0 \\ 0.858 + \frac{5.651}{1 + e^{-24.713\left(\frac{x}{X}\right) + 24.830}}, & 0 < \frac{x}{X} \leq 1 \end{cases}$$

The  $u_p$  distribution in each region generally follows a parabolic distribution, with a tendency for particles to move toward the wall in dilute phase and dense phase regions, while in the transition region, it presents an obvious centripetal particle flow, thus forming a solids internal circulation. Key factors such as particle size, gas velocity, and particle concentration significantly influence the  $u_p$ . Two models of predicting  $u_p$  were established based on physical model derivation (theoretical model) and GA-BP neural network, respectively. Both models demonstrate excellent accuracy, with  $R_2$  values exceeding 0.99. However, the theoretical model has a simpler form with clear physical meaning and a wider range of applications.

This study offers a new perspective and methodology for the quantitative characterization of lateral gas–solid flow in industrial CFB boilers. The established simple models for calculating lateral voidage and particle lateral movement velocity can be integrated into the two-dimensional CFB mathematical model. In the future, we will verify the models in this paper on two-dimensional/three-dimensional CFB test benches of different scales. In combination with combustion reactions, we will further explore the lateral non-uniformity of gas–solid flow, heat transfer, combustion, and pollutant emissions within hot-state CFBs.

#### CRediT authorship contribution statement

**Manxia Shang:** Writing – original draft, Methodology, Investigation. **Ling Jiang:** Investigation. **Li Zhao:** Writing – review & editing. **Zhong Huang:** Writing – review & editing. **Xiwei Ke:** Writing – review & editing, Methodology. **Junfu Lyu:** Writing – review & editing, Funding acquisition.

#### Declaration of competing interest

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

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#### Appendix A. Supplementary data

Supplementary data to this article can be found online at <https://doi.org/10.1016/j.ces.2025.121773>.

#### Data availability

Data will be made available on request.

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